# Hammering Wear Impact Fatigue Hypothesis WEC/irWEA failure mode on roller bearings

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Hansen Transmissions International nv

Kontich Belgium



# Hammering Wear Impact Fatigue Hypothesis

#### Research results

- Thansen communication policy is to release the WEC/irWEA research results in an early stage:
  - stimulate other researchers
  - <u>support</u> maximal the wind industry for gearbox reliability increase

This presentation is based on the <u>actual status</u> of research data and can be reviewed.



Wind Turbine Tribology Seminar

# Hammering Wear Impact Fatigue Hypothesis

# **Agenda**

- ¬1. Introduction
- ¬2. Material observations in WEC/irWEA failed bearings
- ¬3. Interpretation of material observations
- ¬4. Hansen wind experience
- ¬5. Hypothesis development & Material research
- ¬6. Proposals for WEC/irWEA research

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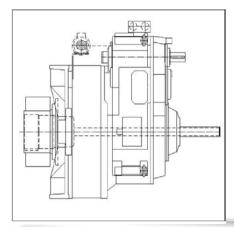
7. Summary and way forward

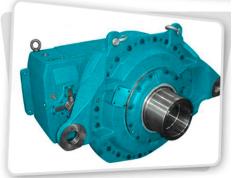


This presentation is revealing research information and industrial solutions to increase the <u>reliability</u> of <u>bearings applied in wind turbine gear units.</u>













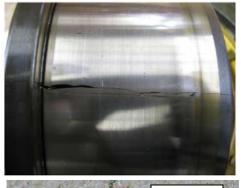


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"Several roller bearing applications are prone to the White Etching Crack (WEC) or irregular White Etching Area (irWEA) failure mode. The subsurface microstructure of a failed bearing contains changed material structures near cracks which are seen as white (and thus chemical inert) after a nital etching test.

Martensitic bearing steel









Bainitic bearing steel



The WEC/irWEA failure mode is observed on several roller bearing applications, with different bearing types from different suppliers.

#### ¬Some examples from the public domain:

- Automotive sector :
  - pulley systems
  - powertrains
- WTG

Wind Turbine Gearbox Reliability

The Nature of the Problem
by Don McVittie
(as channeled by Brian McNiff)



The bearing applications seem to have in common a dynamic operation condition.

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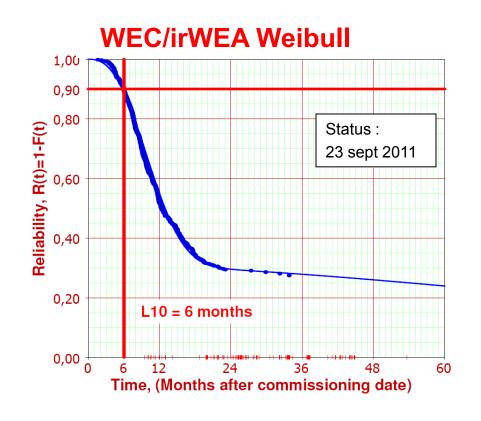
#### ¬Facts:

- calculated lifetime :
  - $L_{10h,ISO}$  = 24 years
  - $L_{10h,FC} > 100 \text{ years}$
- real lifetime L<sub>10</sub> = 6 months

real lifetime is 50 times lower then calculated lifetime

#### ¬Conclusion:

WEC/irWEA ≠ subsurface material fatigue

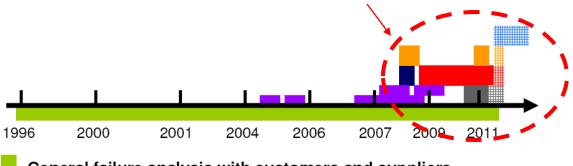


- There is no industry wide agreed understanding of the WEC/irWEA failure mode. The different hypotheses are :
  - overload fatigue
  - hydrogen embrittlement
  - normal stress hypothesis
  - additional load mechanism :
    - mechanical impact
    - electrical
    - chemical
  - •
- In case of a WEC/irWEA bearing failure, best practice is applied.



Wind Turbine Tribology Seminar

The wind industry applications are suffering hard of the WEC/irWEA failure mode on roller bearings. By lack of solutions and understanding from the bearing suppliers, Hansen started independent WEC/irWEA research.



- General failure analysis with customers and suppliers
- Investigation at TU Berlin
- Electron microscopy investigation programs
- Very High Cycle Fatique Test program at Laboratoire Energétique Méchanique Electromagnétisme
- Measurement campaigns
- Material impact tests
- WEC simulator test rig



- Throw area for WEC/irWEA research are the roller bearings in the **Hansen wind gear unit applications**
- The evaluation of:
  - different hypotheses via :
    - load measurement campaings
    - hydrogen measurements
    - material analysis
    - research results
    - . . .
  - Hansen wind experience :
    - identification of drivers
  - ⇒ (hypothesis, formulation



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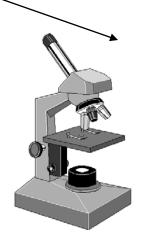
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# Microstructure etching tests :

- cut a sample out of a bearing
- embedding and polishing
- nital etching
- microscopic investigation









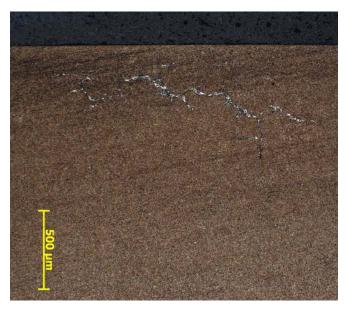


### Case carburised bearing steel: WEC/irWEA failed bearings

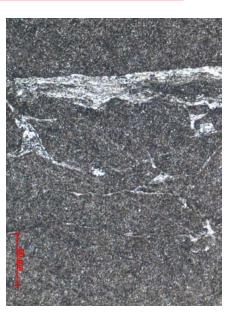
#### Subsurface networks with cracks and white etching area's



Small network



Large network

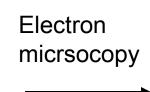


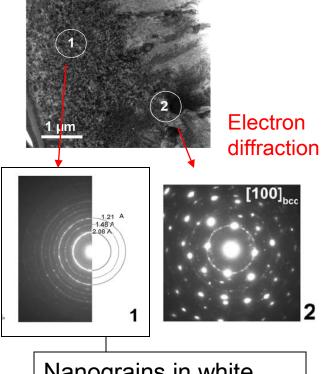
Close-up view

# **Bainitic** bearing steel: WEC/irWEA failed bearings

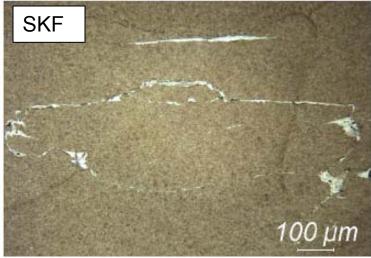
Subsurface networks with cracks and white etching area's







Nanograins in white etching material area's



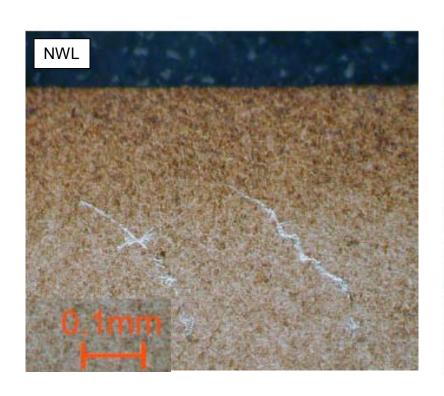
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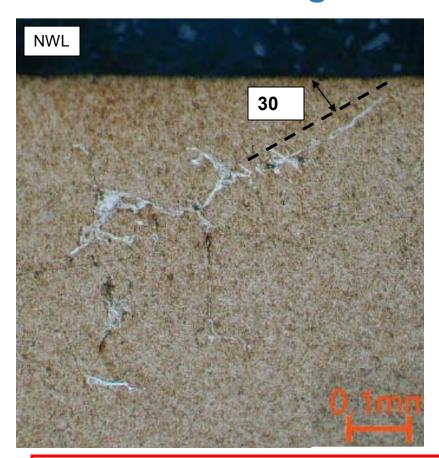
# Martensitic bearing steel: WEC/irWEA failed bearings

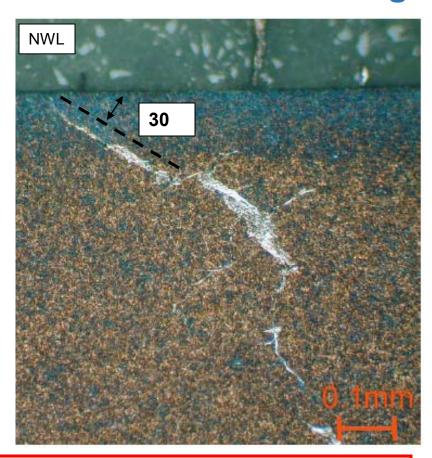




**Isolated small** subsurface damage with white etching area's

## Martensitic bearing steel: WEC/irWEA failed bearings





Subsurface damage with 30 oriented crack towards the raceway and white etching development

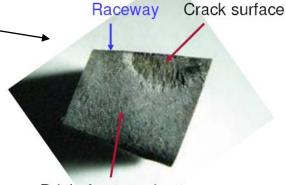


# ¬Crack opening:

- select a small (recent) hairline crack
- cut the sample and open the crack
- investigate crack surface with (electronic) microscopy



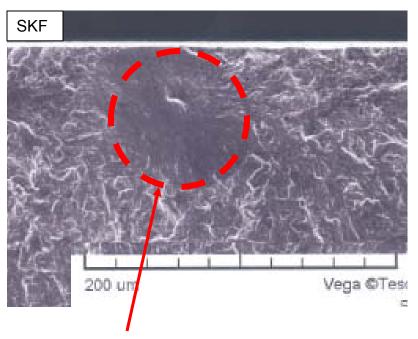




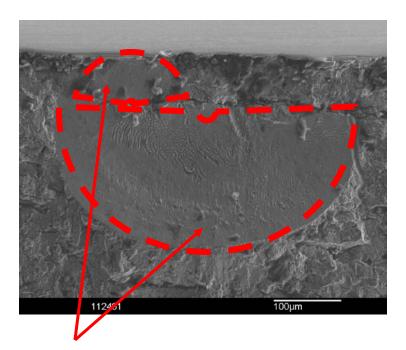
Brittle fracture due to crack opening



# Martensitic bearing steel: WEC/irWEA failed bearings



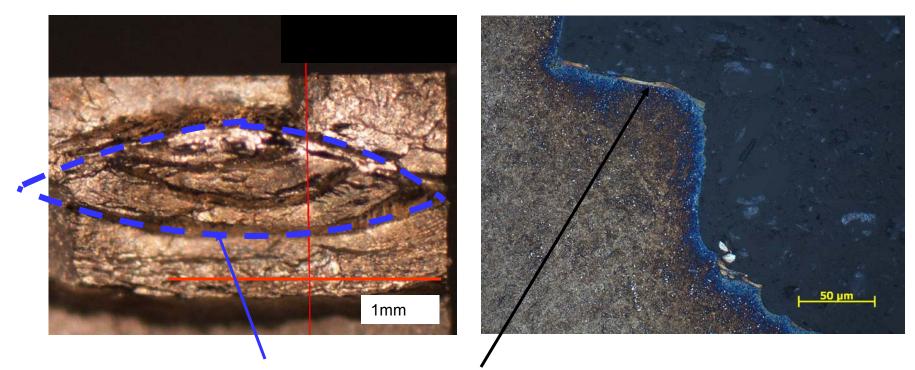
<u>Circular</u> crack around an inclusion/butterfly



Semi circular cracks around an inclusion



# Martensitic bearing steel: WEC/irWEA failed bearings

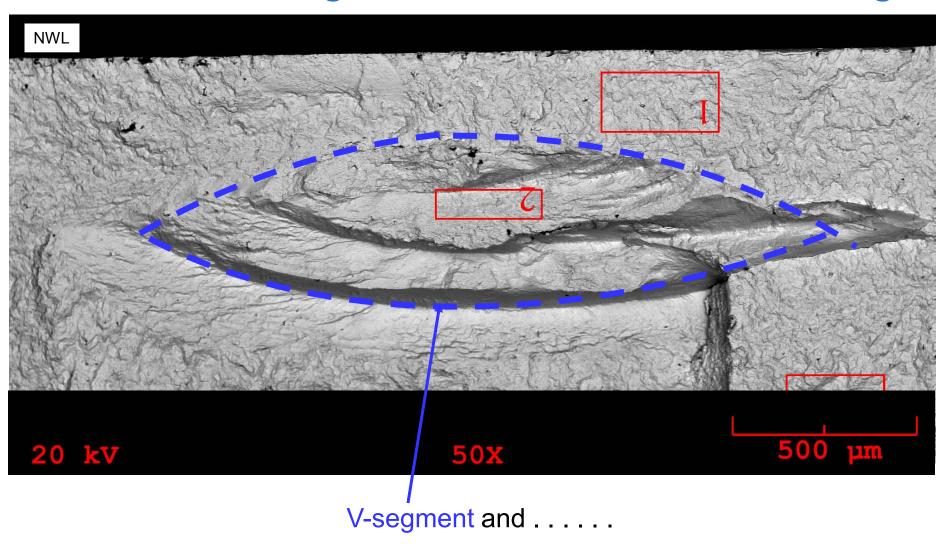


V-segment with white etching material area's in parallel part of the crack

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# Martensitic bearing steel: WEC/irWEA failed bearings

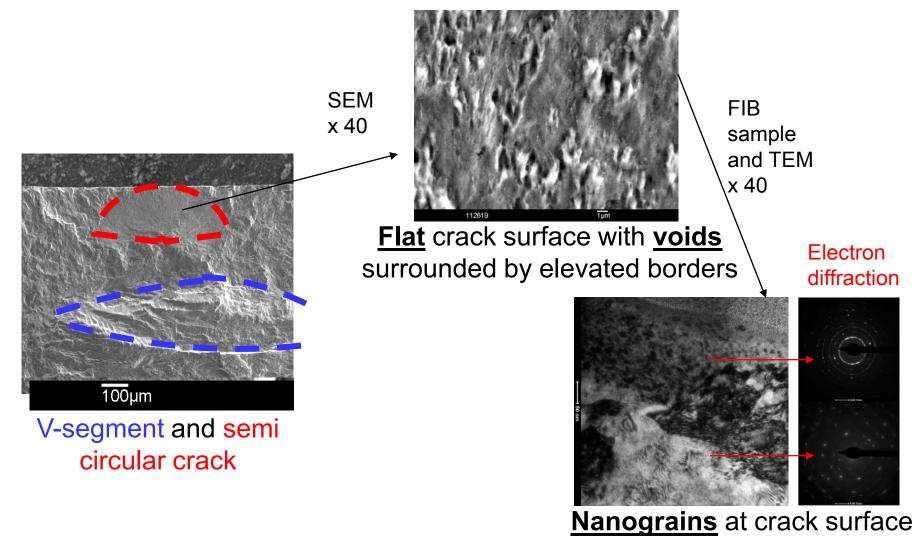


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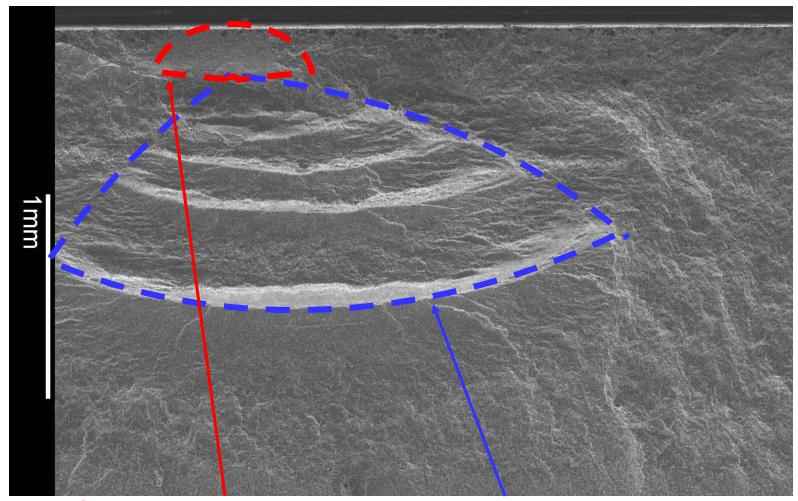


# Martensitic bearing steel: WEC/irWEA failed bearings





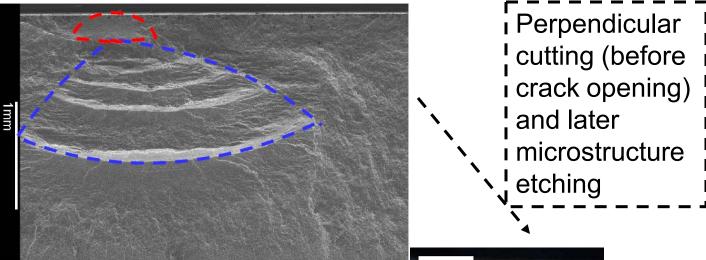
# Martensitic bearing steel: WEC/irWEA failed bearings



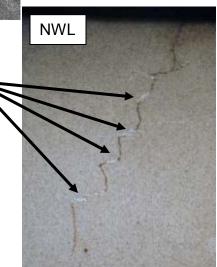
Semi circular crack and <u>large</u> V-segment



# Martensitic bearing steel: WEC/irWEA failed bearings



Semi circular crack and large V-segment with white etching material area's in parallel parts of the crack





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# **Residual stress**

¬ Martensitic : ~ 100 / 150MPa tensile stress

Tensile stress ⇒ (semi) circular crack and V-segment

- ¬ Bainitic : ~ 50 / 100 MPa compressive stress
- Case carburised steel : \_\_\_\_100 / 400 MPacompressive stress

Compressive stress ⇒ large subsurface networks with cracks and white etching area's.



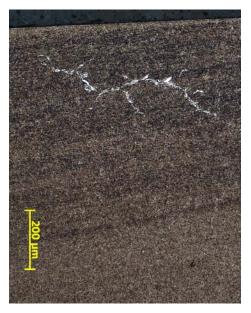


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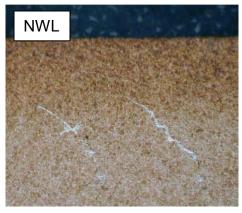
#### ¬Small subsurface damage = local subsurface damage initiation



**Bainite** 



Case carburised



Martensite

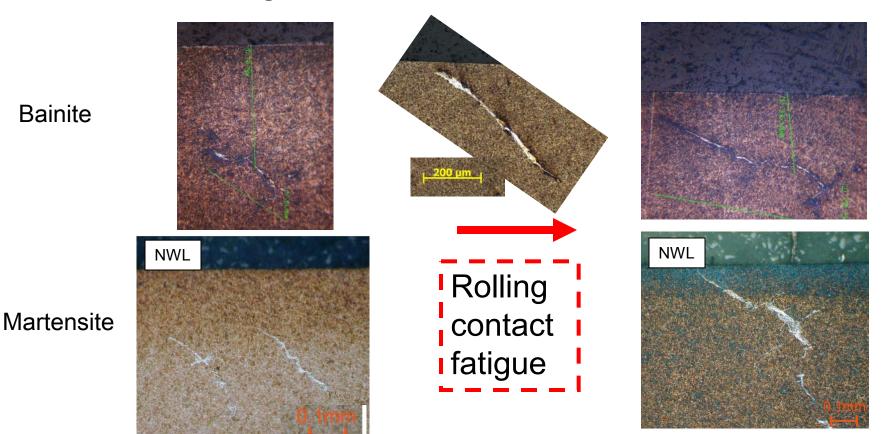








Local subsurface damage initiation = big inclusion. Crack development via classical rolling contact fatigue towards the surface.





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Mat. observation	Severity classification	Interpretation of load system		Mat. interpretation
	Initiation phase, no bearing failure. Limited subsurface damage over about 0.1 – 0.2 mm length.		Severity classification of all mater observation	
	Progression of rolling contact fatigue, just before bearing failure (= raceway damage)			
	Failed bearing with axial hairline crack. Limited subsurface damage till about <b>0.2 mm depth.</b>	Below	v the red ¦ failed	
	Failed bearing with axial hairline crack. Moderate subsurface damage till about 0.7 mm depth.	<b>+</b>		
	Failed bearing with axial hairline crack. Big subsurface damage till about 1.5 mm depth.			



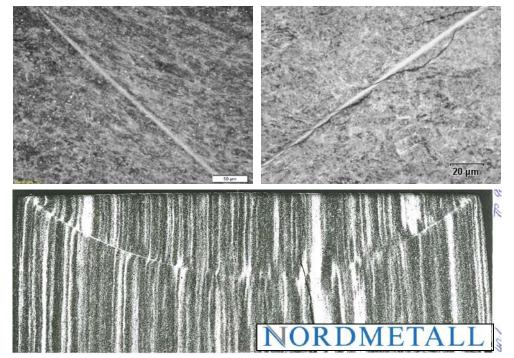
	Т			
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	Failed bearing with axial hairline crack. Big subsurface damage till about 1.5 mm depth.			



Mat. observation	Severity classification	Interpretation of load system	Mat. interpretation
	Initiation phase, no bearing failure. Limited subsurface damage over about 0.1 – 0.2 mm length.	Initiation by an <u>additional load</u> <u>system</u>	
	Progression of rolling contact fatigue, just before bearing failure (= raceway damage)	Initiation by an <u>additional load</u> <u>system</u> (= big inclusion) + rolling contact fatigue	Link severity with load system(s):
	Failed bearing with axial hairline crack. Limited subsurface damage till about <b>0.2 mm depth.</b>	Local weak spot of inclusion and failure development by an <u>additional load system</u> at low level	- <u>additional</u> load system
	Failed bearing with axial hairline crack. Moderate subsurface damage till about 0.7 mm depth.	(Initiation phase +) failure development by an additional load system at moderate level	- RGF
	Failed bearing with axial hairline crack. Big subsurface damage till about 1.5 mm depth.	(Initiation phase +) failure development by an <u>additional</u> <u>load system</u> at heavy level	



- Local subsurface damage initiation :
  - adiabatic shear band (ASB) = straight or curved thin band
  - nano grain sized microstructure, thus white etching
  - generated by an impact load



ASB in bearing steel (upper left and right) and 57HRC steel (lower) generated by an impact test

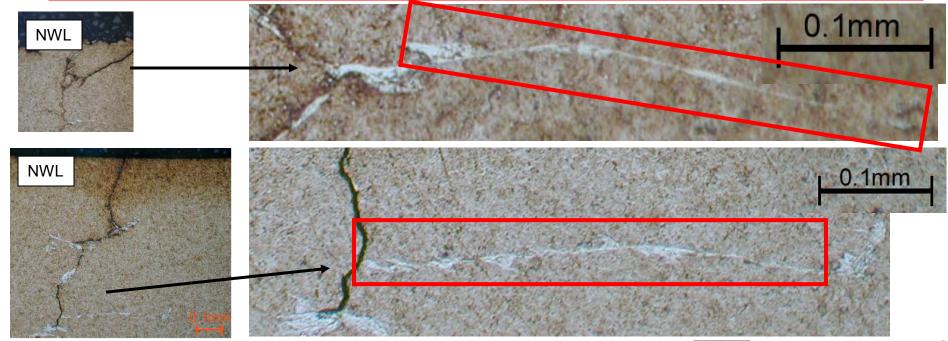
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# White bands are interpreted as adiabatic shear bands



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# ¬Local subsurface damage initiation :

Adiabatic shear band and crack

Rolling contact fatigue

Crack propagation

 Development of white etching area's at the crack and ASB borders

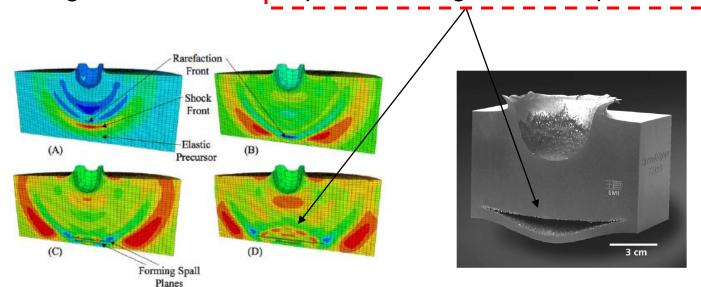
Thus the ASB will be hidden after some time



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#### ¬ Ballistic impact on a plate :

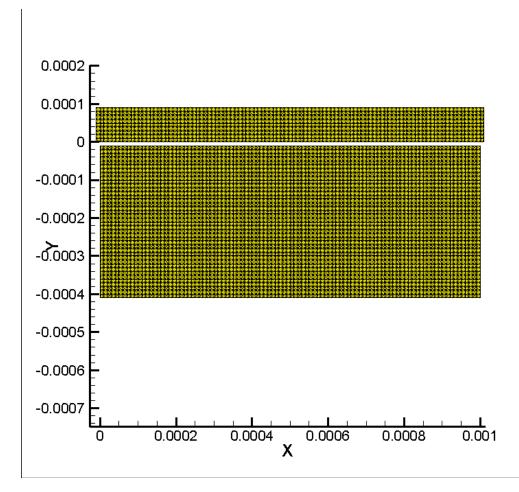
- impact ⇒ generation of longitudinal wave
- superposition of 2 tensile waves :
  - a) reflected wave at free surface = tensile wave
  - b) back side of longitudinal wave = tensile wave
- generation of an elliptical cleavage crack = spallation



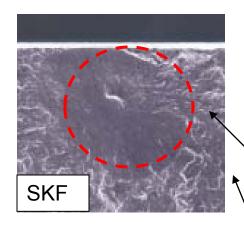
Hiermaier S. J., Structures under crash and impact



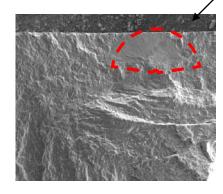
# **Spallation**



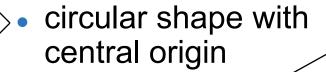
Shaofan Li, http://www.ce.ber keley.edu/~shaof an/spall.html



(Semi) circular crack

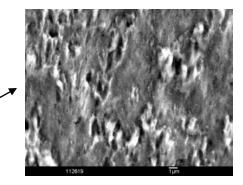


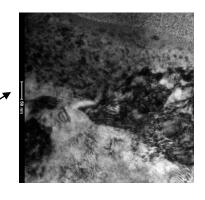
"(Semi) circular crack surface is interpreted as spallation crack from an impact load based on :

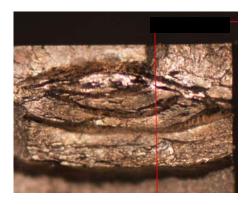




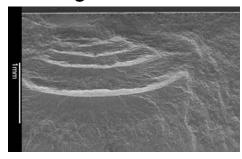
- interaction with inclusions
- presence of nanograins on crack surface







V-segment



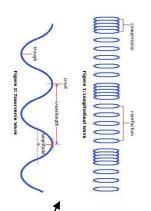


¬V-segment is interpreted as crack surface from a wave load based on :

 V-spreading and stop with depth is propagation of a body wave

 stepped crack pattern: 2 wave systems generate cracks in 2 directions Longitudinal wave 

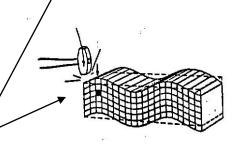
⇒ parallel cracks





Shear wave 

⇒ radial cracks





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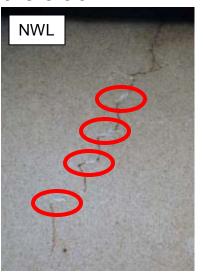
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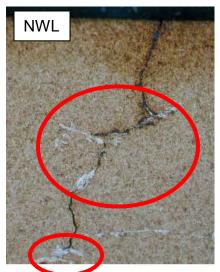
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White etching at parallel part of the crack

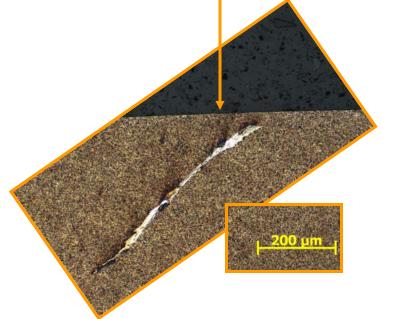




White etching area's at parallel part of crack in V-segment :

 identical generation mechanism as in rolling contact fatigue with crack development

and thus consequential damage



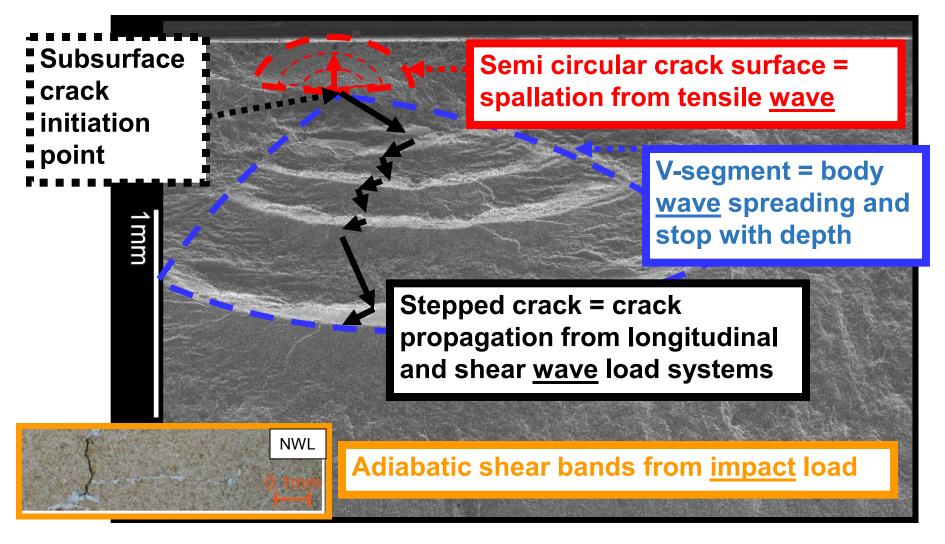
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Research info

#### 3. Interpretation of material observations

#### ¬Martensitic WEC/irWEA failed bearing





Mat. observation	Severity classification	Interpretation of load system	Mat. interpretation
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	Progression of rolling contact fatigue, just before bearing failure (= raceway damage)	Initiation by an <u>additional load</u> <u>system</u> (= big inclusion) + rolling contact fatigue	
	Failed bearing with axial hairline crack. Limited subsurface damage till about <b>0.2 mm depth.</b>	Local weak spot of inclusion and failure development by an <b>additional load system</b> at low level	
	Failed bearing with axial hairline crack. Moderate subsurface damage till about <b>0.7 mm depth.</b>	(Initiation phase +) failure development by an additional load system at moderate level	
			2
	Failed bearing with axial hairline crack. Big subsurface damage till about 1.5 mm depth.	(Initiation phase +) failure development by an additional load system at heavy level	



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Mat. observation	Severity classification	Interpretation of load system	Mat. interpretation
	Initiation phase, no bearing failure. Limited subsurface damage over about 0.1 – 0.2 mm length.	Initiation by an <u>additional load</u> <u>system</u>	Initiation = Adiabtic shear band and crack from an impact load.
	Progression of rolling contact fatigue, just before bearing failure (= raceway damage)	Initiation by an <u>additional load</u> <u>system</u> (= big inclusion) + rolling contact fatigue	White etching = consequential damage that hides the ASB
	Failed bearing with axial hairline crack. Limited subsurface damage till about <b>0.2 mm depth.</b>	Local weak spot of inclusion and failure development by an <u>additional load system</u> at low level	Spallation crack surface from an impact load
	Failed bearing with axial hairline crack. Moderate subsurface damage till about 0.7 mm depth.	(Initiation phase +) failure development by an <u>additional</u> <u>load system</u> at moderate level	Spallation crack surface, V-segment and stepped crack pattern from an impact load  White etching =
	Failed bearing with axial hairline crack. Big subsurface damage till about 1.5 mm depth.	(Initiation phase +) failure development by an <u>additional</u> <u>load system</u> at heavy level	consequential damage



#### Conclusions:

The WEC/irWEA bearing material failure mode

Subsurface impact damage

(and the white etching area's are consequential damage)

The additional load system

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**IMPACT** 



#### Hammering Wear Impact Fatigue Hypothesis

#### **Agenda**

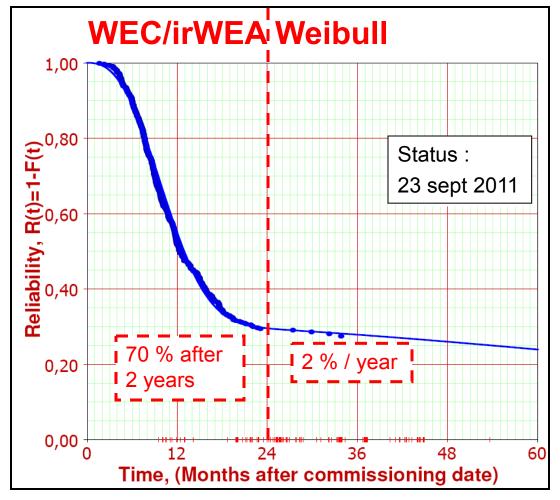
- ¬1. Introduction
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- Promoting the WEC/irWEA bearing failure mode :
  - SRB and BB
  - big roller size
  - high dynamic bearing applications
- Preventing the WEC/irWEA bearing failure mode :
  - "operation time"
  - "case carburised"
  - black oxidised
  - hot assembly



The WEC/irWEA Weibull shows a significant drop of the failure rate <u>after a certain operation time</u>.





Black oxidised performance :

Significant performance increase

Version	Gearbox population	WEC/irWEA faiture rate	After
Standard CRB (bainite & martensite)	1000	<b>/</b> 40 % ↔ 70 %	2 years
Black oxidised CRB (bainite)	1150	0,1 %	2 years

#### ¬ Black oxidised :

- controlled oxidation of the raceway
- iron oxide layer : 1 ↔ 2 µm thick
- improved run in behaviour



#### ¬Case carburised WEC/irWEA performance:

Microstructure	Assembly temperature	Gearbox population	WEC/irWEA failure rate	After
Through hardened martensite	100 C	400	40 %	14 months
Case carburised	100 C	57	5,3 %	11 months
	110 ↔120 C	109	1.8 %	16 months

#### ¬Conclusions:

- case carburised of a certain bearing supplier is <u>an</u> <u>improvement, but not a full robust solution</u>
- preventing influence of increased <u>assembly temperature</u>



Increasedbearingassemblytemperature

Bearing assembly temperature	Gearbox population	WEC/irWEA failure rate	After
100 C	400	40 %	14 months
110 C		12 %	
130 C	34	0 %	35 months
110 ↔120 C	41	7.3 %	15 months

- ¬ Bearing assembly temperature :
  - I WEC/irWEA failure mode is put on / off / on by different lbearing assembly temperatures
  - accurate data



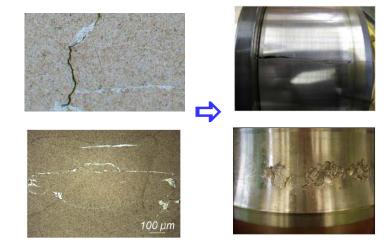
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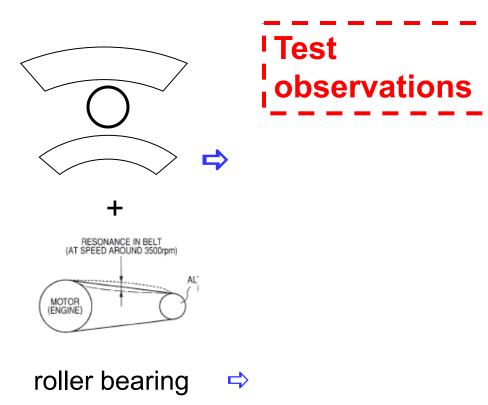


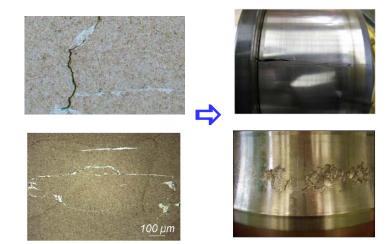
## Interpretation of material observations



⇒ subsurface ⇒ WEC/irWEAimpact bearingdamage failure

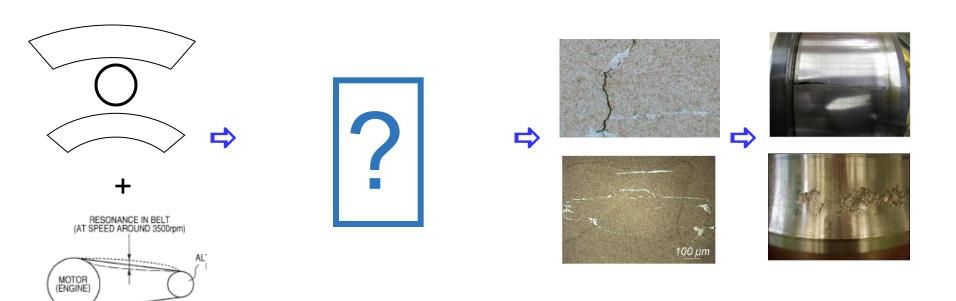
#### **IMPACT LOAD**





WEC/irWEA bearing failure

dynamic load conditions from application



roller bearing
+

dynamic load
conditions
from application

what is the missing link in the chain of events?

subsurface 

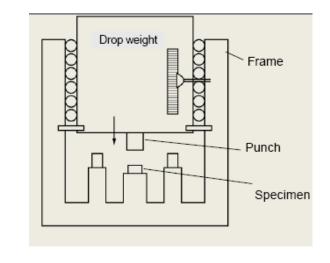
⇒ WEC/irWEA impact bearing damage failure

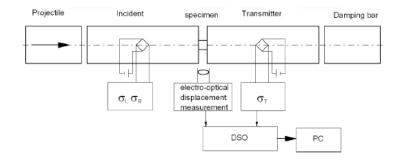
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Hammering Wear Impact Fatigue Hypothesis



#### "Impact tests:

- working hypothesis: CC, BO and HA bearing versions have a higher impact loadability then a standard M or B bearing
- action plan : do impact tests and compare all the variants
- test results :







#### ¬Impact tests:

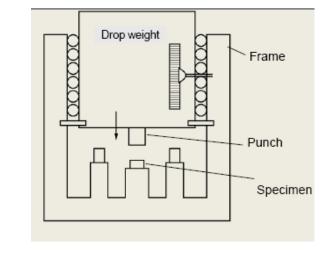
- working hypothesis: CC, BO and HA bearing versions have a higher impact loadability then a standard M or B bearing
- action plan : do impact tests and compare
- test results : low impact loadability but no differentation !!!
- interpretation : CC, BO and HA are better then standard M or B:

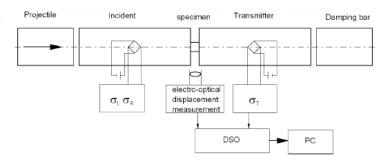


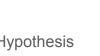




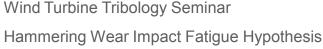






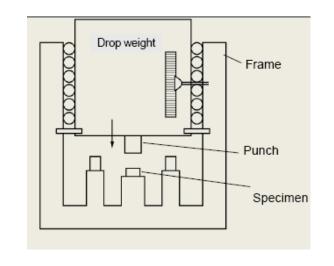


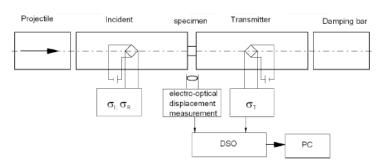




#### ¬Impact tests:

- working hypothesis: CC, BO and HA bearing versions have a higher impact loadability then a standard M or B bearing
- action plan : do impact tests and compare
- test results: low impact loadability but no differentation !!!
- interpretation : CC, BO and HA are better then standard M or B :
  - nigher impact loadability
     no generation of impact loads



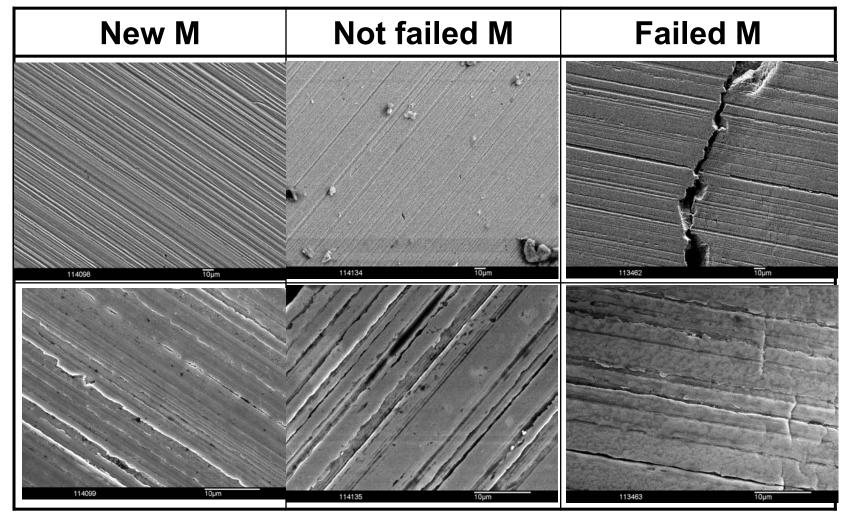




- ¬Preventing drivers:
  - "operation time"
  - "case carburised"
  - black oxidised
  - hot assembly
- Is there a common factor that can be linked with generation of impact loads?
- ¬ Where to look? Previous research ⇒ reduced FWHM at raceway surface!



¬ Comparison of raceway surface : martensitic = M



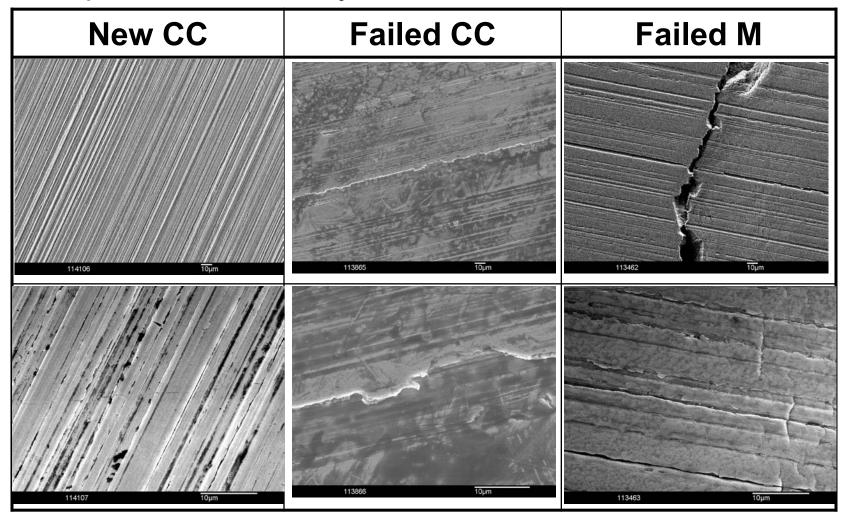
Hammered apperance of raceway surface in a WEC/irWEA critical application.

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Hammering Wear Impact Fatigue Hypothesis



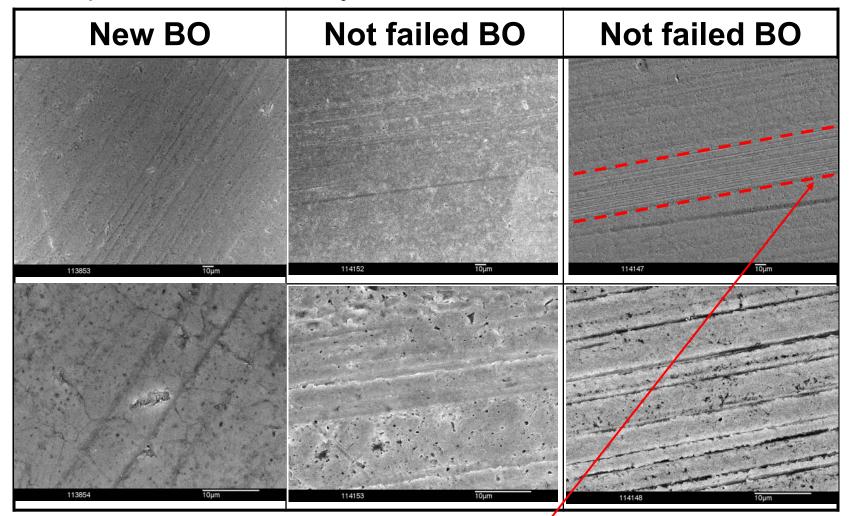
Comparison of raceway surface : case carburised = CC



Hammering apperance on raceway surface is smaller in comparison with martensite. Some case carburised asperities tops have minor flattening.

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Comparison of raceway surface : black oxidised = BO



Black oxidised coating limits the development of hammered apperance to

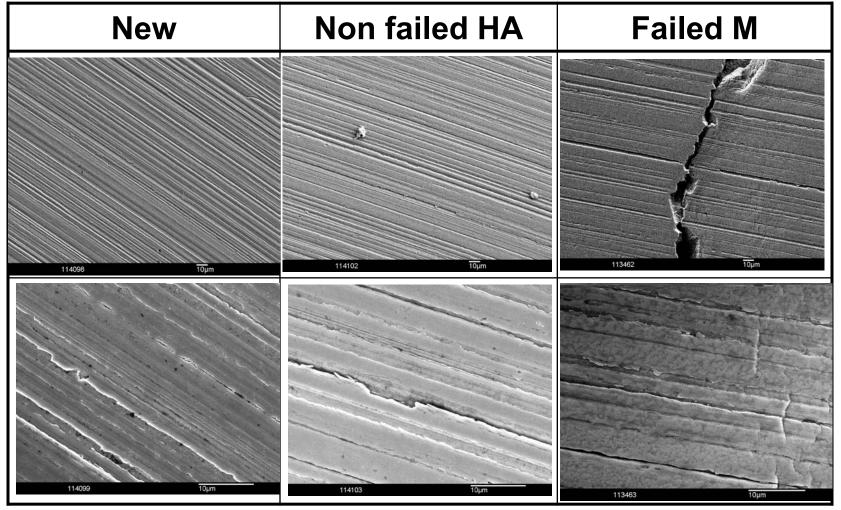
isolated lines or small bands

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Hammering Wear Impact Fatigue Hypothesis



Comparison of raceway surface : assembly at 130 C = HA



Visual appearance is less hammering for HA compared to M.

¬ Comparison of raceway surface roughness

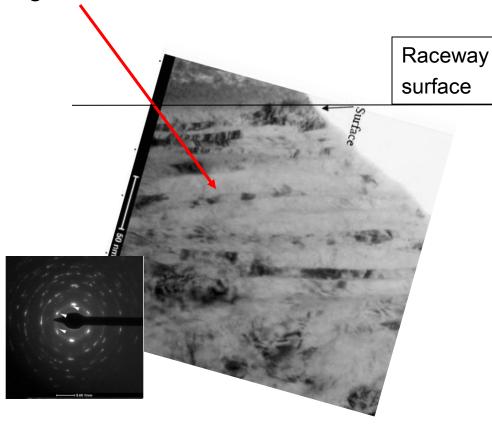
Variant	Ra new raceway		Ra loaded	draceway
	(µm)		(µm)	
M	0.090	0.009	0.055	0.003
Standard martensite	$\Delta = 0.035$			
HA	0.092	0.007	0.079	0.011
Bearing assembly at 130 C		$\Delta = 0$ .	.013	

Hammering of HA raceway surface is **smaller** then standard martensite.



### ¬Why less hammering with a HA bearing?

¬New bearing raceway surface ⇒ 100/200 nm layer of small lamellar grains about 20/30 nm thick



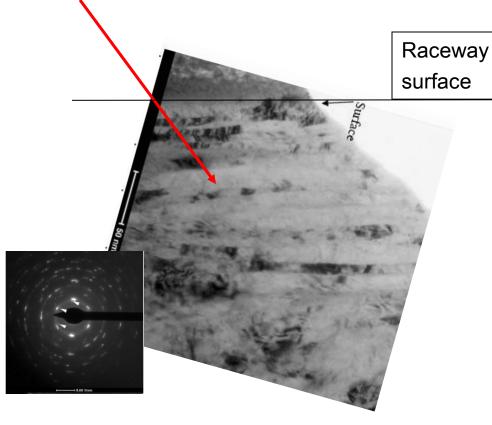


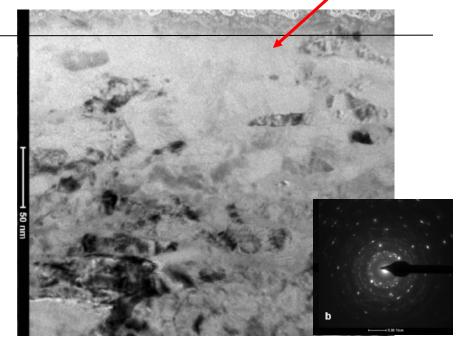
## ¬Why less hammering with a HA bearing?

¬New bearing raceway surface ⇒ 100/200 nm layer of small lamellar grains about 20/30 nm thick

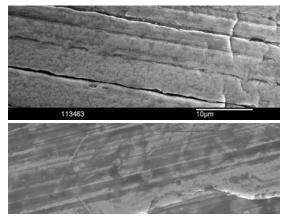
¬After bearing assembly at 130 C

⇒ 100/150 nm recrystallized layer at raceway surface with grains of about 5/10 x 30/40 nm

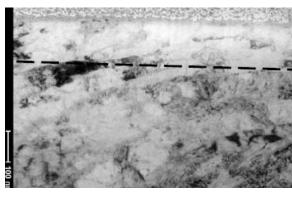












# ¬Preventing drivers:

- "operation time":
   hammering = strain
   hardening ⇒ after
   some time
- "case carburised": retained austenite
- black oxidised : —— oxide layer
- hot assembly : recrystallized layer

Common factor = raceway wear resistance

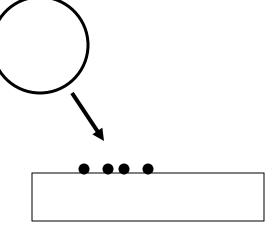




¬Is there a link between wear and generation of impact load?

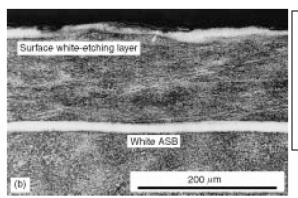
#### ¬Experiment:

- oblique ball impact on a steel plate
- quartz particles on the plate





#### Test result:



B. Zhang, Adiabatic shear bands in impact wear

#### ¬Conclusion:

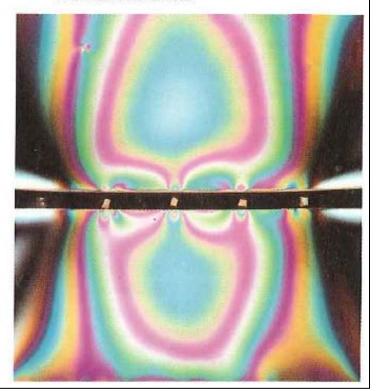
- load system of "ball drop + <u>particle</u>" ⇒ subsurface impact damage
- load system :
  - ball drop = limited impact load
  - particle on the surface = impact load concentrator



#### Particles in EHD contact under static load = stress risers

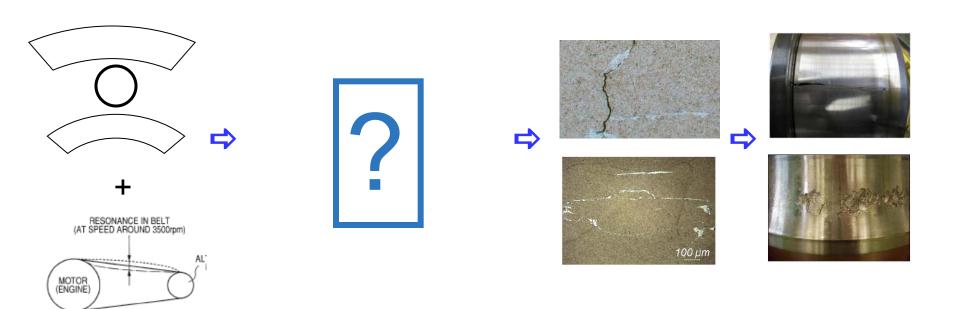
Fig 2 Optical interference patterns showing stress in a model of a rolling contact

- a) clean surfaces completely separated by lubricant film
- b) solid contaminant particles contained in the lubricant film



Wuttkowski, loannides, The effect of contaminants on bearing life.



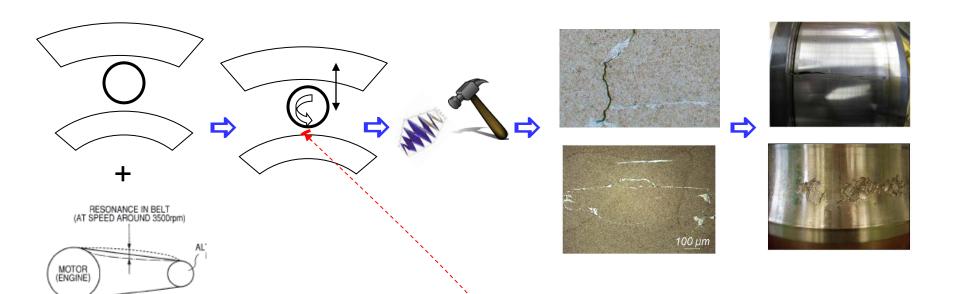


roller bearing dynamic load conditions from application

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what is the missing link in the chain of events? IMPACT LOAD subsurface WEC/irWEA impact bearing failure damage





roller bearing +
dynamic load
conditions

from application

raceway hammering = & shear stress

= hammering

wear particle

- + rolling contact

subsurface impact damage WEC/irWEAbearingfailure

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¬What is the mechanism behind raceway hammering?

#### ¬Possibilities:

end of EHD pressure peak

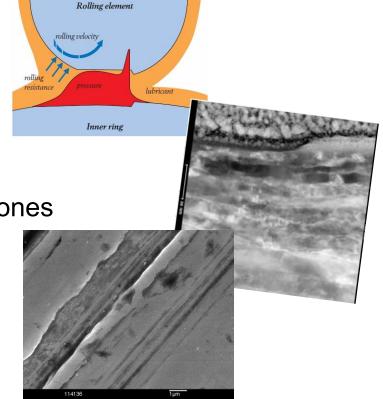
mixed friction

yes: hammering in dedicated load zones

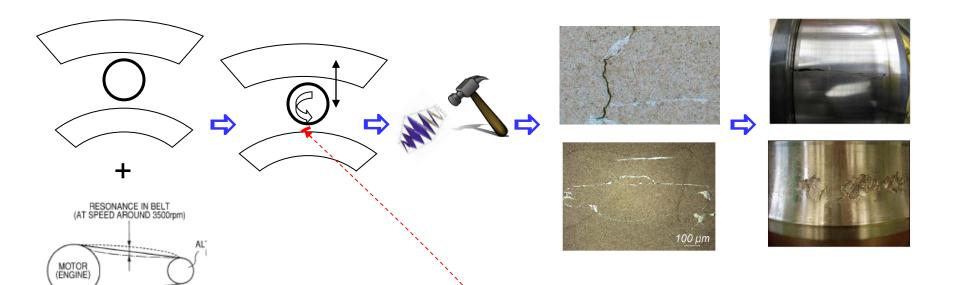
 no : no signs of material removal at the raceway surface

• . . .

¬Answer: to be investigated







roller bearing +

dynamic load
conditions
from application

raceway hammering

& shear stress

= hammering

wear particle+ rolling contact

subsurface impact damage WEC/irWEAbearingfailure

Potential to enter via other wear driving mechanisms

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## Promoting the WEC/irWEA bearing failure mode :

- SRB and BB : internal slip
- big roller size :
  - higher mass
  - higher inertia ⇒ increased slip

high dynamic bearing applications

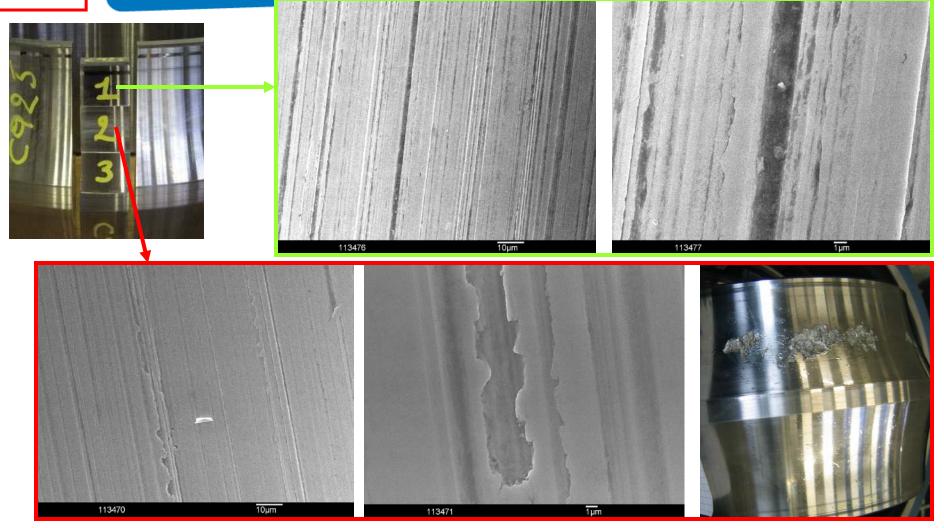


Contributors towards hammering wear and shear stress



Research info

### 5. Hypothesis development & Material research



Raceway of inner ring SRB of WEC/irWEA citical bearing application (not failed). More intense hammering wear on sample 2 from middle of raceway where spalling is initiated in case of WEC/irWEA bearing failure.

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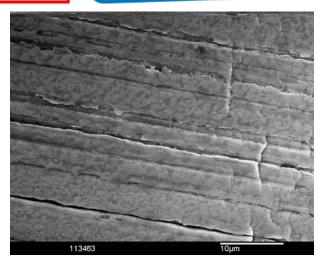
16 nov 2011

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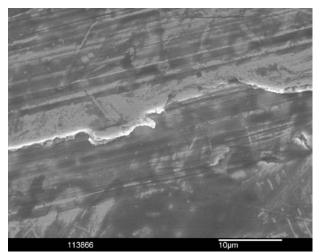
Hammering Wear Impact Fatigue Hypothesis



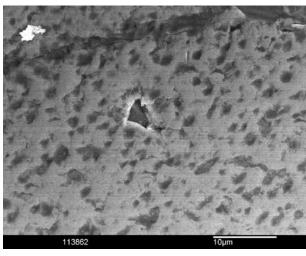
### 5. Hypothesis development & Material research



Raceway martensite bearing supplier



Raceway case carburised bearing supplier



Raceway case carburised Hansen

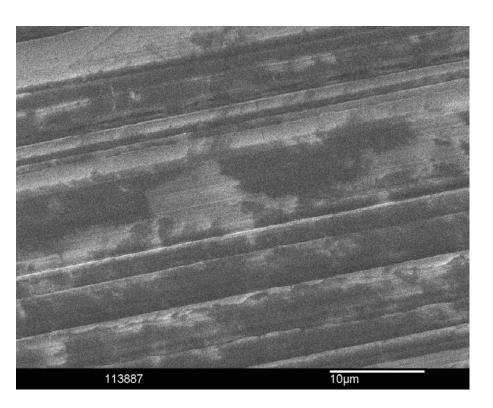
Comparison between raceways. Lower picture is Hansen case carburised without hammering wear. No WEC/irWEA failures on Hansen case carburised

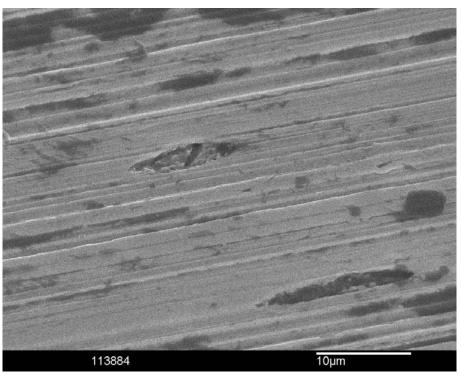
raceways.

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Research info

### 5. Hypothesis development & Material research





Hansen high speed gear tooth with limited (left) and more (right) wear development. Gear flank topography has minor hammering wear of asperities tops. No WEC/irWEA gear failures observed on Hansen gears.

### Hammering Wear Impact Fatigue Hypothesis

# **Agenda**

- ¬1. Introduction
- ¬2. Material observations in WEC/irWEA failed bearings
- ¬3. Interpretation of material observations
- ¬4. Hansen wind experience
- ¬5. Hypothesis development & Material research
- 76. Proposals for WEC/irWEA research
- 7. Summary and way forward



#### 6. Proposals for WEC/irWEA research

"BACK TO BASICS.

- "What are the basic technology area's?
- ¬Material damage in failed bearings ⇒ IMPACT
- ¬EHD contact ⇒ TRIBOLOGY
- The identified counter measures are interpreted as surface treatments that increase the wear resistance ⇒ WEAR and SURFACE TREATMENTS



### 6. Proposals for WEC/irWEA research

# ¬ Impact :

- simulate impact load system (http://www.ce.berkeley.edu/~shaofan/spall.html):
  - simulate material damage
  - determination of load system
  - hammering wear hypothesis
  - surface treatments
  - microstructures
  - . . . .

#### experimental

conceptual test rig to simulate material failure mode :

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- ball impact
- wet plate with different roughness values & coatings

(see exisiting test rigs for coatings)

+ high frequent (> 1 MHz) instrumentation



### 6. Proposals for WEC/irWEA research

```
¬ Tribolgy :
```

- theoretical work :
  - model micro EHD
- experimental work on small scale (fundamental research) :
  - measure pressure peaks :
    - roller element impact
    - flat wear particle on (coated) surface
    - contaminated oil
  - measure wear
  - . . . . .
- experimental work on large scale (validation) :
  - large bearing test rig + dynamic load conditions
  - large gear unit test rig + dynamic load conditions



# Hammering Wear Impact Fatigue Hypothesis

# **Agenda**

- ¬1. Introduction
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- ¬5. Hypothesis development & Material research
- ¬6. Proposals for WEC/irWEA research

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7. Summary and way forward



- ¬Material observations in WEC/irWEA
- ⇒ bearing material failure mode = subsurface impact damage
- Subsurface crack initiation point

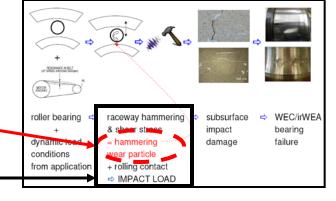
  Semi circular crack surface = spallation from tensile wave

  V-segment = body wave spreading and stop with depth

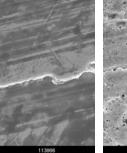
  Stepped crack = crack propagation from longitudinal and shear wave load systems

  NWL

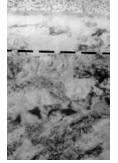
  Adiabatic shear bands from impact load
- The root cause hypothesis is reviewed
  - hammering wear is a link in the chain of events
  - raceway wear particle + rolling contact = IMPACT LOAD



Increased raceway wear resistance is common tribological system of the industrial solutions









### "WEC/irWEA bearing failure mode in gear driven WTG:

- big size roller bearings + vibrations ⇒ reliability risk
- control the risk!

#### THansen strategy:

- share the know how in an early phase
- increase confidence in the gear driven WTG concept



- The identified robustness increasing measures are :
  - raceway surface treatment
    - black oxidised
    - recrystallized layer by Hansen hot assembly
  - case carburised microstructure
    - Hansen case carburised microstructure (zero WEC/irWEA failures)
    - inconsistency between bearing suppliers (preferred bearing supplier)

Hansen is introducing this for new gearbox designs and serial production

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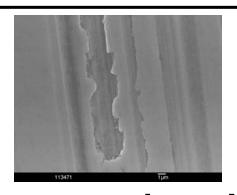


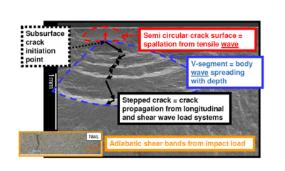
- Truther research by **bearing suppliers** of the **new** impact fatigue failure mode on bearings :
  - complete understanding
  - cost effective solutions
  - application dependant solutions

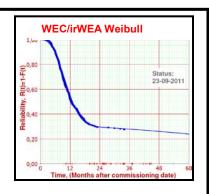


Research info

### Hammering Wear Impact Fatigue Hypothesis







Hammering Wear Impact **Hypothesis** 

Fatigue

We believe that we can control the risk

Thank you for your attention

Questions?





Research info

#### **How to increase the WEC/irWEA robustness?**

- "What can be effective as WEC/irWEA robust increasing counter measures ?" ⇒ "How can we avoid or reduce raceway wear ?"
  - (reduce vibrations)
  - wear resistance raceway surface :
    - recrystallized layer via Hansen hot assembly
    - strain hardening via shotpeening, rolling, . . .
    - hard coatings: nitrogen, . . .
  - wear resistance microstructures
    - case carburised (reference towards Hansen and preferred supplier)
    - microstructures with austenite: nanostructured bainite, ...
    - nitrogen charged microstructures

#### optimized EHD contact to avoid or limit wear :

- soft coatings:
  - black oxidised (stabilized wear conditions)
- lubricant & oil film:
  - grease (sealed bearing)
  - additives (increased friction and or wear is a risk)
  - reduce raceway surface roughness



Apply the most

cost effective

solution